

# Combined System and Performance Trade-Off Autotuning for PID Controllers

Jerwin Prabu A<sup>1\*</sup>, Ashish Tupate<sup>1</sup>, Jenifer S<sup>2</sup> and Jackulin S<sup>3</sup>

<sup>1</sup>Department of Robotics Research and Development, Bharati Robotic Systems India Pvt Ltd, Pune, India

<sup>2</sup>Department of Electronics and Communication Engineering, Francis Xavier College of Engineering, Tirunelveli, India

<sup>3</sup>Department of Electronics and Telecommunication Engineering, Karpagam College of Engineering, Coimbatore, India

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\*Corresponding author: Jerwin Prabu A, Department of Robotics Research and Development, Bharati Robotic Systems India Pvt Ltd, Pune, India, E-mail: jerwinprabu@gmail.com

## Abstract

Control system has been widely used in various fields. The angle error and turning angles are the main factors that affect the accuracy performance of the interfacing system. It is important to notice that these control loop parameters error cannot be eliminated from the interaction system completely. Within these possibilities, the control system's performance and the robustness must be considered as important. The aim is to provide solutions and improve the general behavior of a control system, with a Single-Degree-of-Freedom (1DoF) Proportional-Integral-Derivative (PID) controller structure. Even after calibration, these errors still exist and will be fluctuated during the operating system running. This paper proposed a new method of finding the best position and orientation for turning to perform a specific working path based on the current accuracy capacity of the control loop system. By analyzing the system forward/inverse kinematics and the angle error sensitivity of different combination in the PID system, a new evaluation formulation is established for combined operation and performance trade-off autotuning methods for PID controllers. The accomplishment of the claimed robustness is checked. The influence of different position and orientation on the movement accuracy of the end effector has been verified by experiments and discussed thoroughly.

**Keywords:** PID control; Trade-off; Tuning methods; Optimality degree; Regulatory-control; Parametric stability;

## Introduction

Business Proportional - Integrative - Derivative (PID) controllers have been with no uncertainty the broadest alternative that can be found in modern control applications. This reality influences PID to control less demanding to comprehend by the system operators than other most progressive system strategies. Uncommon consideration is made of the workshop – all-encompassing of PID Control, where a look at the best in class on PID control was given [11]. In addition, in view of the across the board utilization of PID controllers, it is fascinating to have straightforward yet productive strategies for tuning the controller. Considering that in mechanical process control applications, it is required a decent load-unsettling influence dismissal (generally known as administrative control), and also,

a great transient reaction to target changes (It can be defined as angle control system), the process control configuration ought to think about the two potential outcomes of operation. Many techniques for this kind of PID controllers have been figured in the part of the most recent years (Sung SW, et al. [9]; Sudha & Deepa, [16]), and furthermore some specific utilization of the 2-DoF detailing in view of cutting-edge advancement calculations have been produced. Regardless of the over, the servo and direction requests can't be all the while happy with a Single-Degree-of-Freedom (1-DoF) controller, in light of the fact that the subsequent for every performance mode is extraordinary and it is conceivable to pick only an ideal arrangement. The proposed techniques consider 1-DoF PID system; it is an elective when express double-Degree-of-Freedom (2-DoF) PID system is not accessible. Thusly, it can be expressed that the proposed model can be utilized when multi-operation performance may happen and it can be viewed as an understood 2-DoF process control system (on the grounds that the plan considers the two targets, servo and direction modes). In this model, the variety of the established different control Margin measures to a solitary and broader evaluation of vigor; for example, Considering the significance of the express consideration of heartiness into the plan, the point is to search for an ideal tuning for a joined servo/direction list that likewise ensures a vigor esteem, determined as an attractive Maximum Sensitivity prerequisite.

## Related Work

As of late, tuning techniques in view of advancement approaches with the point of guaranteeing hearty soundness have gotten consideration in the writing (Joelianto E, [10]). Likewise, extraordinary advances on ideal techniques in view of settling PID arrangements have been accomplished (Prabu AJ, et al. [18]). However, these strategies, albeit powerful, use to depend on to some degree complex numerical enhancement systems and don't give auto-tuning regulation. Rather, the tuning of the process control is characterized as the arrangement of the streamlining issue. In addition, now and again the strategies considered just the framework execution (Chen D & Seborg DE,

[4]), or its heartiness (Tuning, [2]; Bascetta L and Leva A, [6]; Fung, et al. [3]). In any case, the most intriguing cases are the ones that consolidate execution and power, since they look with every one of framework’s necessities (Narmadha J and Prabu AJ [12]). The past referred to strategies think about the execution and strength together in the control plan [14,17]. Be that as it may, nobody treats particularly the execution/strength exchange off issue, nor consider in the definition the servo/direction exchange off or the communicating between these factors.

### Implementation of Proposed Model

We consider the solidarity criticism framework appeared in figure 1, where System is the systematic and Measuring element is the (1-DoF) process controller. The factors of intrigue could be portrayed as takes after:

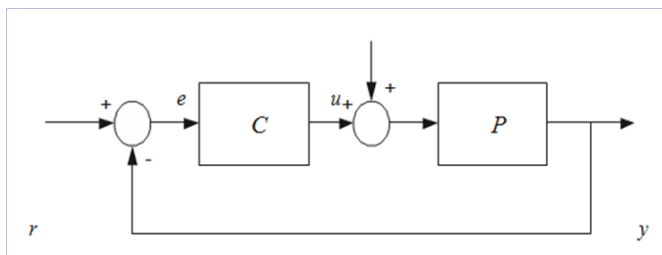


Figure 1: Feedback Systematic Process control system [11].

This model is usually utilized as a part of process control on the grounds that is basic and portrays the elements of numerous modern procedures roughly (Shamsuzzoha M and Lee M, [11]). The accessibility of system in the process control function is a verifiable truth. The age of such system simply requirements for an exceptionally basic advance test trial to be connected to the procedure. This can be considered as leverage regarding different techniques that need a more plant requesting analysis, for example, strategies in view of more perplexing models or even information-driven strategies where an adequately rich info should be connected to the plant. Starting here of point, to keep up the requirement for site experimentation is an actual moment that thinking about the modern use of a system appeared in figure 1. In this unique circumstance, a typical portrayal of the particular procedure control estimation is done as far as the standardized dead-time  $\tau = L/T$ . Where  $K_p$  is the corresponding addition,  $T_i$  is the indispensable time steady and  $T_d$  is the subsidiary time consistent. The subordinate time commotion channel consistent  $N$ , as a rule, takes esteems inside the range 5-33. Without loss of sweeping statement, here we will consider  $N = 10$ . The subject of the control framework arrangement appeared in figure 1 where the process control PC(s) takes the unequivocal type of a 1-DoF closed loop system PID controller. At the point when the configuration for ideal end-point (Process-control) reaction is viewed as the controller requirements are balanced. Note that because of the fitting method, the tuning articulations do exclude the entire scope of  $\tau$ , in this manner split in two, bringing about various constants for every parameter. The use of the ISE auto-tuning analysis for ideal end-point and load unsettling influence reactions gives the PID parameters appeared in table 1 and figure 2.

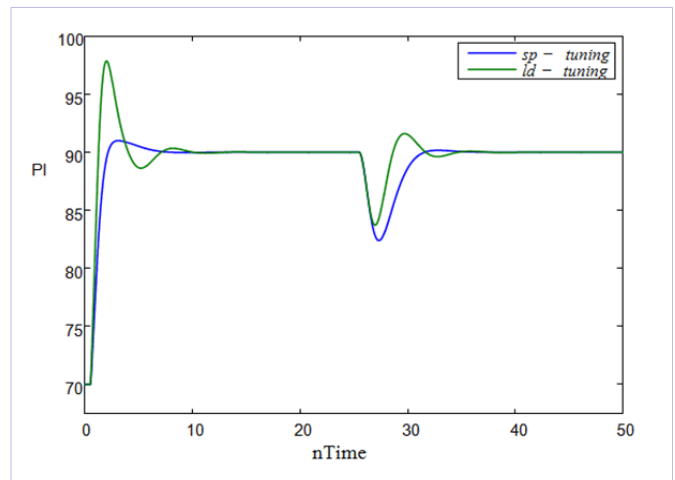


Figure 2: Execution example - Responses for servo access control and regulation for control system PI.

Table 1: PID tuning range settings for (SP) and (LD)

$\tau$ range	0.5 - 1.5		1.5 - 2	
Tuning	EP	LD	EP	LD
a1	1.436	1.492	1.97	1.924
b1	-0.67	-0.966	-0.76	-0.735
a2	1.342	1.125	1.54	1.767
b2	-0.765	-0.753	-0.33	-0.921
a3	0.892	0.975	1.68	1.561
b3	0.995	0.998	0.56	0.763
a1	1.436	1.492	1.97	1.924
b1	-0.67	-0.966	-0.76	-0.735

It demonstrates the execution of the two function when the system control process is working in both, system and direction mode. In this manner, the watched Performance Degradation is bigger for the table 2. From a worldwide perspective, it will appear to be smarter to pick the set-point settings.

Table 2: P1 Load-Disturbance tuning

Auto-tuning	$K_p$	$T_i$	$T_d$
end - point(sp)	1.767	1.351	0.513
load - disturbance(ld)	2.638	1.012	0.559
Control tuning variation	1.563	0.994	1.735

The execution of the control framework is estimated regarding an execution file that considers the likelihood of an operation mode unique in relation to the chosen one. Where  $x$  means the working method of the system control framework and chose working parameters for auto-tuning, i.e., the control tuning mode.

$$J_{sp}(sp) \leq J_{sp}(ld) \quad (1)$$

$$J_{ld}(ld) \leq J_{ld}(sp) \quad (2)$$

Subsequently, we have determined  $x \in \{sp, ld\}$  and  $z \in \{sp, ld\}$ ,  $sp$  represents for end-point (servo control system) auto-tuning and  $ld$  for stack unsettling influence (controller) tuning.

Execution won't be ideal for the two circumstances. The Performance in control system measure in the assessment of the abundant of execution regarding their ideal esteem. Note that, on the grounds that the controller settings communicated through unequivocal reliance on the procedure standardized dead-time  $\tau$ , it merits considering that, for the control system application, the Performance in control system will likewise rely upon  $\tau$ . Note likewise that Performance in control system is a diminishing capacity of the standardized releasing-time. An ultimate conclusion for the suitable auto-tuning mode will rely upon the significance of the framework operation as control modes. Be that as it may, we might want to exploit existing auto tuning formulae, with a specific end goal to keep the system, and the subsequent controller articulation, in also straightforward terms. In this manner, the subsequent process control requirements could be mentioned as an augmentation of the ideal ones. On this premise we characterize a process control parameterized as far as a vector as

$$\gamma = [\gamma_1, \gamma_2, \gamma_3] \quad (3)$$

The qualities of this factor are limited to  $\gamma \in [0, 1.5]$   $I = 1, 2.5, 3$ . Additionally, the end-point autotuning will relate a shape imperative for each  $\gamma_i = 0$ , while the heap unsettling influence tuning compares to  $\gamma_i = 1.5$ . The controller settings family  $[K_p(\gamma_1), T_i(\gamma_2), T_d(\gamma_3)]$ , can be communicated, in a more broad shape, as

$$\begin{aligned} K_p(\gamma_1) &= fK_p(\gamma_1; K_{p,l_d}, K_{p,sp}) \\ T_i(\gamma_2) &= fT_i(\gamma_2; T_{i,l_d}, T_{i,sp}) \\ T_d(\gamma_3) &= fT_d(\gamma_3; T_{d,l_d}, T_{d,sp}) \end{aligned} \quad (4)$$

where  $\gamma_i \in [0, 1.5]$   $I = 1, 2.5, 3$  and  $[K_{p,sp}, T_{i,sp}, T_{d,sp}]$  and  $[K_{p,l_d}, T_{i,l_d}, T_{d,l_d}]$  remain for the end-point and process system-unsettling influence settings for  $[K_p, T_i, T_d]$  individually. Additionally, every  $\gamma_i$  change needs to fulfill the shape imperatives with the frame

$$T_{i,l_d} = fT_i(1; T_{i,l_d}, T_{i,sp}) \quad (5)$$

$$T_{d,sp} = fT_d(0; T_{d,l_d}, T_{d,sp})$$

$$T_{d,l_d} = fT_d(1; T_{d,l_d}, T_{d,sp}) \quad (6)$$

Taking (5) as the shape imperatives detailing, the process control parameters can be produced by a straight advancement of the requirements from the end-direct auto-tuning toward the heap aggravation. Hence, it can be determined as,

$$\begin{aligned} T_i(\gamma_2) &= \gamma_2 T_{i,l_d} + (1 - \gamma_2) T_{i,sp} \\ T_d(\gamma_3) &= \gamma_3 T_{d,l_d} + (1 - \gamma_3) T_{d,sp} \end{aligned} \quad (7)$$

Here, the goal is to present the steadiness examination of the shut circle produced by the controller characterized by (7) regarding the vector  $\gamma$ . In the first place, consider that the PID controller is communicated with its three picks up as,

$$K_c = K_p, \quad K_i = K_p / T_i, \quad K_d = K_p T_d \quad (8)$$

The transitional controller given by (7) asymptotically balances out the framework gave that the outskirts esteems are given and table 2,  $1/N$  is adequately little and  $\gamma_i \in [0, 1.5]$  for  $I = 1, 2.5, 3$ . It means presented above, we will demonstrate that the proposed fringe esteems for  $K_c$ , in particular,  $K_{c1}, K_{c2}$ , ensuring

then the presence of a settling PID controller. For each estimation of  $\tau \in [0.5, 1.5] \cup [1.5, 2]$ , figure 3 demonstrates the most extreme permitted corresponding addition and alternate increases given by the tuning conditions for the fringe parameters.

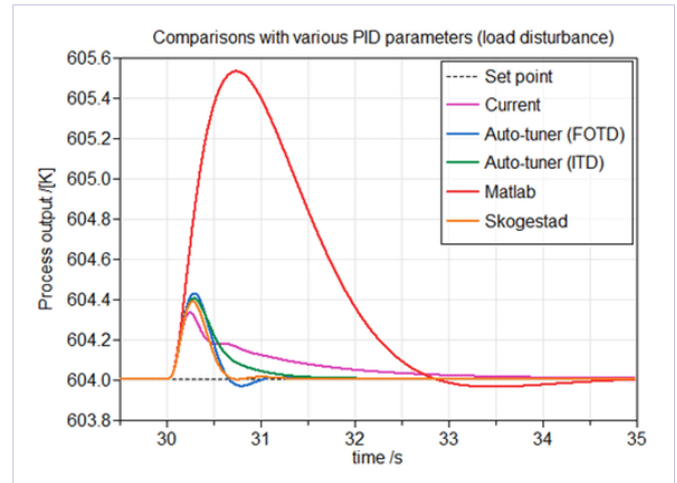


Figure 3: Proportional gain and slope values obtained by the auto-tuning parameters [7].

The evidence is finished delineating the rest of the system control parameter variety and checking that the plotted polygon exists at the crossing point. Subsequently, the hypothesis is demonstrated for the perfect PID controller [7]. At long last, for an adequately little  $1/N > 0$ , the shut circle is as yet constant by processing the Tikhonov's Theorem to the subsequent uniquely bothered framework, consequently closing the confirmation of the Theorem. Figure 4 demonstrates the subsequent balancing out district R and the variety of  $K_i$  and  $K_d$  parameters for all  $\gamma$  set in the vicinity of 0 and 1.5.

There might be distinctive approaches to characterize the PD ( $\gamma$ ) work, contingent upon the significance related to each working segment (e.g. applying particular force elements to every individual module) [2]. That can be defined as Weighted Performance Degradation (WPD) list, where  $\alpha \in [0, 1.5]$  is the weight parameter and shows one of the two conceivable operation control modes is favored or more essential. For instance, a framework that works the 75% of the time as a controller (or the other way around 25% as a servo) [8],  $\alpha = 0.75$ . Be that as it may,  $\alpha$  parameter permits to settle on a more broad decision for the inclination of the framework operation (not just considering the ideal opportunity for every operation mode) [3]. Note additionally that with  $\alpha = 0.50$ , speaks to an equal articulation to the one acquired that gives a similar essentialness for considered operation modes. The middle auto-tuning will be controlled by legitimate determination of  $\gamma = [\gamma_1, \gamma_2, \gamma_3]$ .

### Optimization Problem Formulation

Heartiness is a vital quality for control frameworks, on the grounds that the outline techniques are typically in light of the utilization of low-arrange direct models recognized at the shut circle conventional point. Due to the greater part of the mechanical procedure, it is important the normal modification in the process control qualities accepting solidness edges, or vigor necessities,

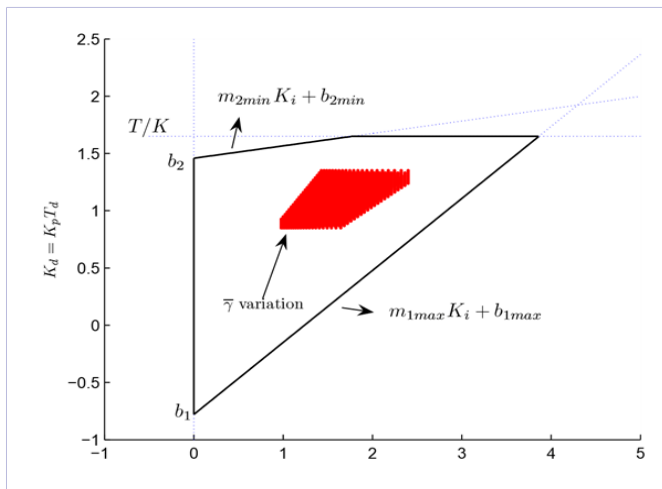


Figure 4: The stabilizing portion of the system and the polygon for the axial  $\gamma$  variation [2].

for the control framework [13,5]. Consequently, the framework execution to stack aggravation and point-guide modification and its heartiness toward a variety of the process control procedure attributes, safeguarding the outstanding exchange off between every one of these factors. Considering the above proclamations, consider the input control system framework appeared in figure 5, where  $R(s)$  is the random control procedure,  $L(s)$  is the loop control,  $e(s)$  is the end-point,  $f(s)$  is the process control flag,  $h(s)$  is the heap aggravation and  $y(s)$  is the framework yield.

where  $n(s) = e(s) - y(s)$  is the blunder,  $K_p$  is the process in static

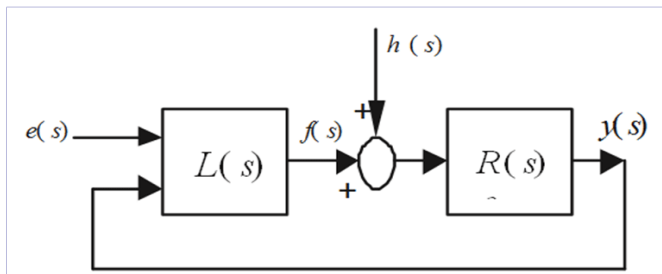


Figure 5: The stabilizing portion of the system and the polygon for the axial  $\gamma$  variation.

pick up,  $T_i$  the indispensable time consistent,  $T_d$  the subsidiary steady and the subordinate channel consistent  $N$  is taken  $N = 10$  as it is regular states in the modern process controller. As a sign of the framework power (relative security) [1], the Sensitivity Function top esteem will be utilized. The end-point control framework huge amount of Sensitivity is characterized.

In this figure 6,  $J^r$  on the vertical axis indicates the evaluation of the reference response, and  $J^d$  on the horizontal axis is that of the disturbance response. Moreover,  $J^{*r}$  denotes the evaluation on the reference response optimization design, and  $J^{*d}$  denotes the evaluation on the disturbance response optimization design.  $M_s$  is ordinarily in the range 1.4 - 2. Consequently, to guarantee  $M_s = 2.0$  gives what is ordinarily viewed as least heartiness necessity (that means  $A_m > 2$  and  $\phi_m > 290$ , for  $M_s = 1.4$  we have  $A_m > 3.5$  and  $\phi_m > 410$ ). It can be seen that the point is the “perfect”

one since it speaks to the base execution esteems considered both conceivable operation, servo, and direction, into the portion. Be that as it may, this is inaccessible process due to the distinctions in the flow for each of progressive goals of the control system functions. In this way, our endeavors must go towards on process finding the base coming about separation, which means the best harmony between the control system functions. On thusly, a target work is planned so as to get nearer, however much as could be expected, the subsequent point, to the “ideal” one. In figure 6, the record is spoken to by the bolt between the “perfect” point and the comparing to the middle of the road auto-tuning. Considering the above examination, the improvement issue points consider the system standardized dead-zone,  $\tau$ , in the processing range  $0.5 \leq \tau \leq 2.0$ , to get the PID control system ideal parameters with the end goal. In system streamlining is finished utilizing hereditary calculations procedure. The point of minimization is to accomplish an adjusted execution for both operation methods of the control framework. The deviation of the subsequent estimation of  $M_s$  as for the predefined target has an immediate impact (as an exchange off) in the execution of the framework. Here, the subsequent heartiness, applying the proposed technique, when contrasted with the coveted one, keeping in mind the end goal to check the achievement of the guaranteed power. Figure 7 demonstrates systematic powerful auto-tuning has a decent precision for the  $M_s$  defined coefficients for all the scope of procedures, subsequently guaranteeing the execution is the exponential one that can be accomplished for the strength esteem.

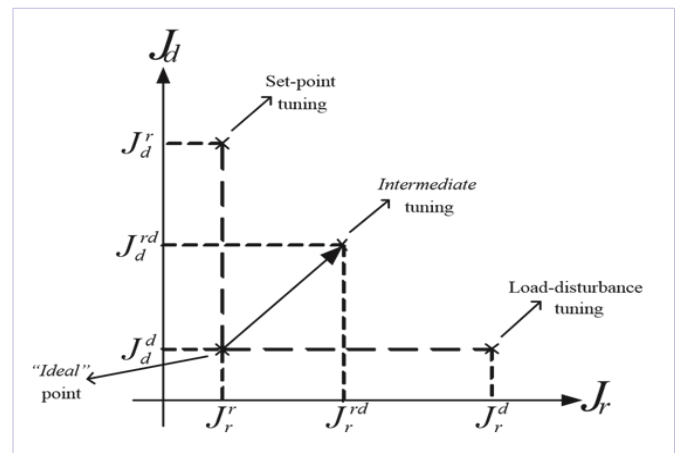


Figure 6: Plane  $J_r - J_d$

From the exceptionally understood execution vigor exchange off, the expansion of the framework’s heartiness from the without msd case (no strength limitation), is reflected in a disintegration of the framework’s execution, and the other way around. Like figure 7, where it can be seen the heartiness expanding, in figure 8 it is demonstrated how the execution file  $J_{rd}$  fluctuates, for every last one of the proposed power levels. On the off chance that we utilize the data of figure 7 and figure 8, and the pre-processing case as the beginning stage, that is conceivable to notice that for each level. The strength is enhanced accomplishing littler esteems for  $M_s$ , yet in the meantime having bigger esteems for the execution list  $J_r^d$ . The connection to the sequence of execution and

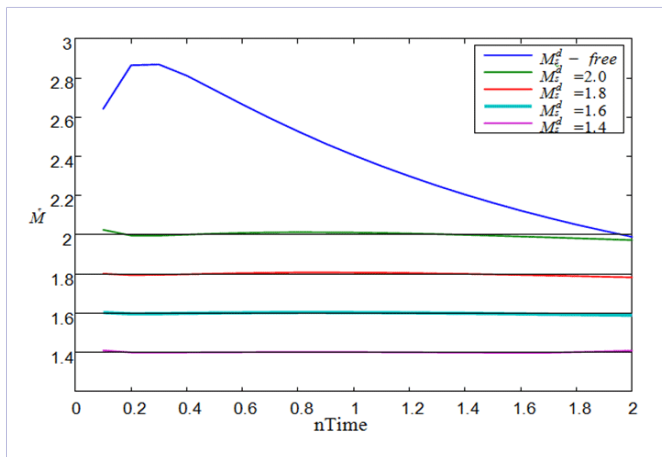


Figure 7: The accomplishment of claimed robustness level.

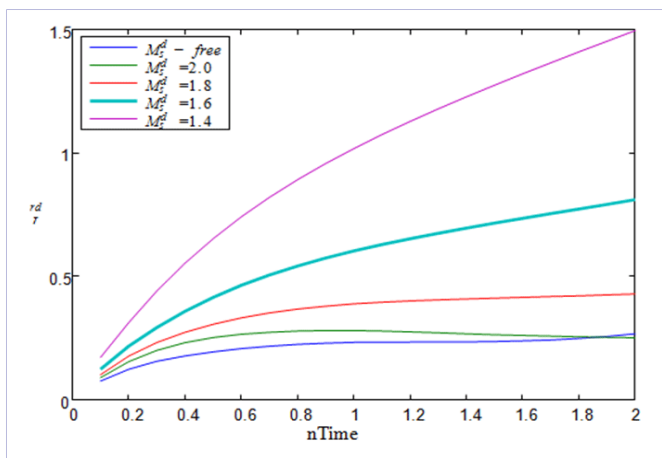


Figure 8: Combined index  $J_r^d$  for each robustness level tuning

the heartiness increment (for a particular level of  $M_s$ ) is improper in terms of the linear system model, neither for a  $\tau$  run.

For instance, in figure 8 specific contrast to the execution for stages  $M_s^d = 1.7$  and  $M_s^d = 1.5$ , is less, in spite of that the levels are similarly isolated. For the normalized control system, the normalized PID parameters optimized subject to the stability margin constraint, in which  $M_s^d = 1.4, 1.6, 1.8,$  and  $2.0$ .

### Result and Discussion

In process control output displayed in figure 9, it is critical to promise some level of heartiness, so as to save the shut circle progression, to conceivable varieties in the control framework. Likewise, in the meantime, it can be given the best execution to linear control and direction.

The majority of the above particulars, prompt have distinctive exchange offs, amongst execution and strength or amongst servo and control modes that must be comprehended in an adjusted path. So, searched for a PID process control auto-tuning decide appearance to the general issue. Figure 10 and figure 11 determines the tuning is ideal, however much as could reasonably be expected, to a proposed execution file that considers both framework operation modes, including additionally a specific level of power, determined as an alluring Maximum Sensitivity

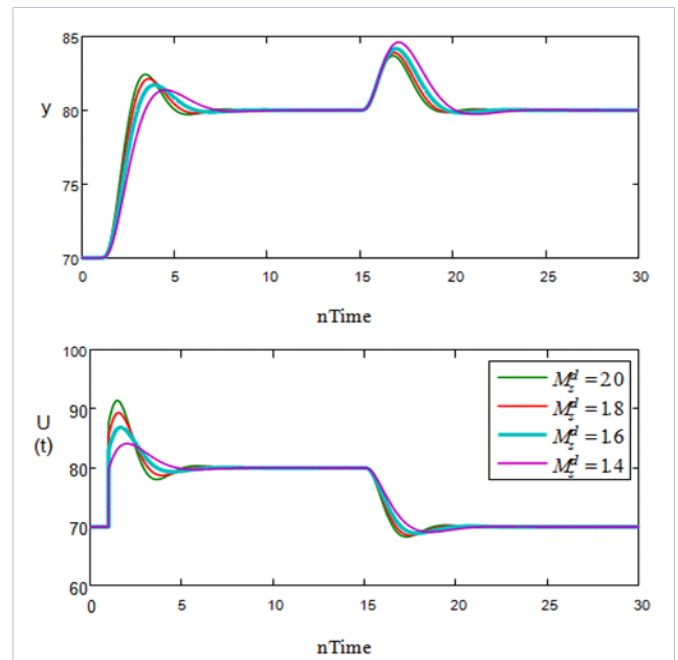


Figure 9: Process Control - Linear method ( $\sigma = 0.50$ ) [3].

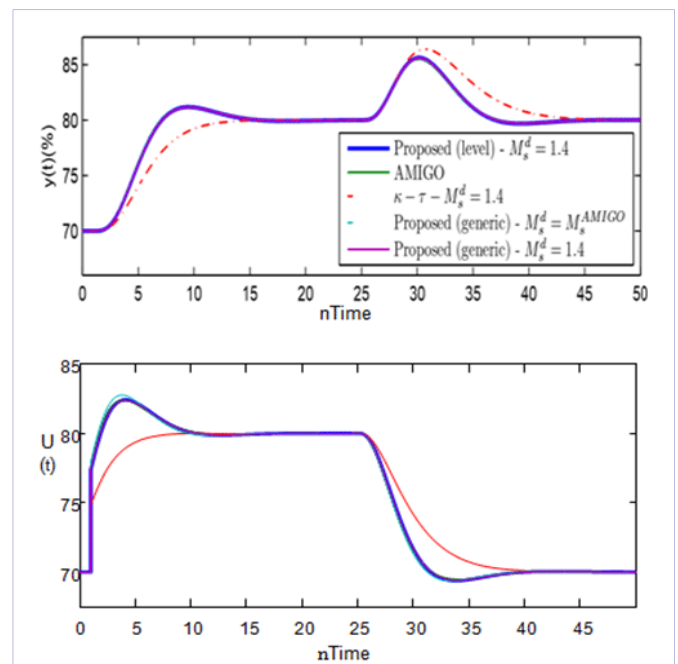


Figure 10: Process control - Generic response and debasement case ( $\sigma = 1.6589$ )

esteem. Auto-tuning formulae have been displayed for two methodologies. To begin with, vigour is set up utilizing a subjective levels grouping and after that, the thought is stretched out to an issue that offers a bland articulation, to permit the particular as far as any estimation of strength in the range  $M_s \in [1.4, 2.0]$ . In addition, considering the execution/power exchange off, the precision of the guaranteed strength is a point that has been checked, accomplishing level bends for the subsequent esteems [15]. To put it plainly, both methodologies are two of the primary

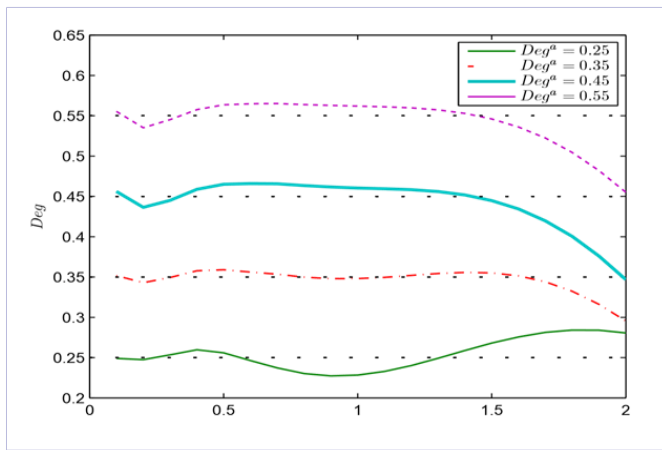


Figure 11: The accomplishment of the fixed degradation level tuning [4].

commitments exhibited in this examination article. In this way, the combination for every control parameter and every corruption stages, are experimented to fit the comparing conditions frame.

Note additionally that, to keep similar tuning articulations gives significantly greater consistency and effortlessness to the proposed approach. Indeed, even the exactness between the accomplished debasement and they chose in  $Deg^a$  (Various acceleration degrees for PID tuning), isn't on a part with the one acquired for the response, it can state that the auto-tuning achievement is adequate, particularly for the principal  $\tau$  and  $\sigma$  values go [0.1, 2]. For  $\tau > 1.3$ , the difference is because of progressive variations for the control system parameters and furthermore, it is a locale of procedures are hard to stabilizing and utilizing a control since it speaks to frameworks ruled by the dead time.

Since the approach depends on the data gave by the settled debasement levels, the scope of legitimacy is inside  $Deg^a \in [0.25, 0.55]$ . It is imperative to underline that, this expansion is simply conceivable in light of the effortlessness and the homogeneity of the PID tuning parameters, that remaining parts a similar articulation for every single one of the settled debasement levels. By and by, the point is to give a non-exclusive detailing keeping in mind the end goal to give culmination, however much as could be expected. Accordingly, in light of the fact that every controller parameter has a similar shape, we search for a general condition as,

$$p_i(\tau, Deg^a) = a_i (Deg^a) \tau^{bi(Deg^a)} + c_i (Deg^a) \quad (9)$$

With the convergence focuses on the comparing sets of bends, we can decide the reasonable arrangement of wanted vigor that gives the best harmony between execution optimality degree and strength increment. Stochastic optimization is used with random (noisy) function measurements or random inputs in the search process. This arrangement decides the strength profile for the best vigor/execution trade-off, for all the  $\tau$  run.

Figure 12 demonstrates the  $I_{Rob}$  variation, as a function, for a few instances for the standardized dead-zone. The PID controller is used to minimize the error signals, or we can define more

rigorously, in the term of error criteria: to minimize the value of performance indices. Index parameters fixed for machine to minimize the error signal. Each system line speaks to the covariant strength increment to a specific procedure (nTime of  $\sigma$ ). Reasonable variation noted in figure 12, it can characterize a specific esteem (stages), then the power increment and simultaneously the crossing point focuses, decide a reasonable heartiness. Auto-tuning is the tuning parameter it gives the trade-off adjusts to the power increment and the subsequent nonlinearity of proportional degree for the framework's execution.

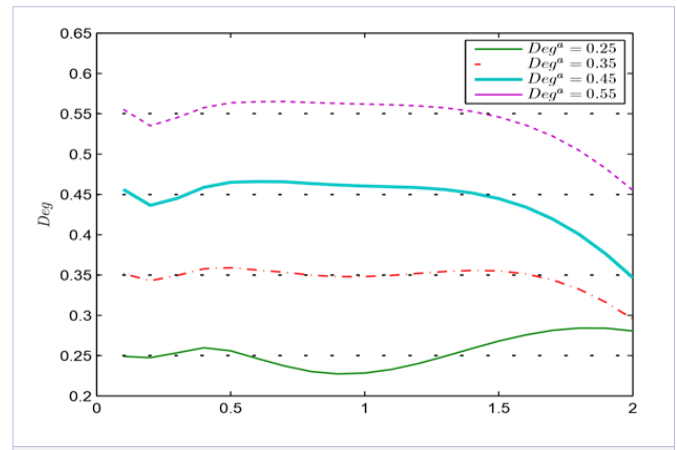


Figure 12: Variation of the index  $I_{Rob}$ .

### Conclusion

New autotuning strategies for PID controllers have been exhibited. On the off chance that the process control has dependably to work on anyone conceivable operation strategy process (servo or control) the auto-tuning decision can be a new process and clear. Be that as it may, when the two circumstances are probably going to happen, it may not be so clear which are the most proper controller settings. A few assessments demonstrate the great outcomes of the proposed tunings contrasted and other understood PID strategies. At long last, it was attractive to search for a middle of the road, it gives a harmony to the level of linearity control and the vigor increment, an all-around adjusted PID tuning was exhibited. An imperative part of another control part is that all PID process control auto-tuning strategies were parameterized utilizing a similar shape for the conditions of controller's parameters. This permits to keep up basic and homogeneous articulations.

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